NASA TECHNICAL MEMORANDUM

NASA TM-82391

AN EVALUATION OF GREASE TYPE BALL BEARING LUBRICANTS OPERATING IN VARIOUS ENVIRONMENTS

(Status Report No. 5)

(NASA-TM-82391) AN EVALUATION OF GREASE
TYPE BALL BEARING LUBRICANTS OPERATING IN
VARIOUS ENVIRONMENTS Status Report No. 5
(NASA) 24 p HC A02/MF A01 CSCL 131

N81-16475

Unclas G3/37 41286

By E. L. McMurtrey Materials and Processes Laboratory

January 1981



NASA

George C. Marshall Space Flight Center Marshall Space Flight Center, Alabama

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U.S. DEPARTMENT OF COMMERCE
NATIONAL TECHNICAL
INFORMATION SERVICE
SPRINGFIELD, VA 22161

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	REPORT NO. NASA TM-82391	2. GOVERNMENT AC		3. RECIPIENT'S CA	
	TITLE AND SUBTITLE An Evaluation of Grease Typ Operating in Various Environ			5. REPORT DATE January 198 6. PERFORMING OR	
	AUTHOR(S) E. L. McMurtrey			8. PERFORMING ORG	ANIZATION REPORT #
9.	PERFORMING ORGANIZATION NAME AND AD	DRESS		10. WORK UNIT NO.	
	George C. Marshall Space Fl Marshall Space Flight Center		812	11. CONTRACT OR G	
12	SPONSORING AGENCY NAME AND ADDRESS			13. TYPE OF REPOR	& PERIOD COVERED
	National Aeronautics and Spa Washington, D.C. 20546	ace Administrat	ion	Technical Me	
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ACKNOWLEDGMENTS

The author wishes to thank S. B. Prestwood and C. A. Torstenson for their work on this project.

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TECHNICAL MEMORANDUM

AN EVALUATION OF GREASE TYPE BALL BEARING LUBRICANTS OPERATING IN VARIOUS ENVIRONMENTS (Status Report No. 5)

I. INTRODUCTION

This is the fifth in a series of status reports to be issued covering a long-term test program to evaluate a number of fluid lubricants in ball bearings operating under various environmental conditions. A previous report [1] discussed the general test program and gave the results of the first series of vacuum ambient temperature tests. Since that report, sufficient progress has been made to provide a comparison of many of the greases being evaluated for ball bearing lubricants in different environments; therefore, it is believed that the information also contained in reports Nos. 2, 3, and 4 [2,3,4] will prove useful to those responsible for selecting lubricants for various space missions.

This program is an extension and expansion of pioneering work done by Young et al. [5] on fluid lubricated bearings operating in vacuum. Because many of the spacecraft planned for the future will require mechanisms that can operate for long periods of time in adverse environments, it is necessary to define the operating limits of available lubricants in these environments. As of December 1980, 400 sets of 800 bearings have completed 1 year of testing, and 100 sets of 200 bearings are undergoing tests. The present plan is to continue the test program using commercially available greases to determine statistically which lubricants will provide maximum bearing operating life with the environmental conditions under which they may be used. This procedure has been used to eliminate all but four candidate lubricants for 5-year tests. These lubricants are now being tested under selected environmental conditions to failure or for a 5-year period.

H. TEST EQUIPMENT

To provide a statistical sample of a number of lubricants operating under various environmental conditions, it is necessary to conduct a large number of tests simultaneously. Therefore, 20 test motors, each containing two test bearings, are set up in each chamber. Each test set consists of four samples (eight bearings) of five different lubricants for the 1 year tests. One test set is shown in Figure 1. The bearings chosen for testing are size R 4, 0.635 cm 1.D. by 1.59 cm O.D. (0.25 in.

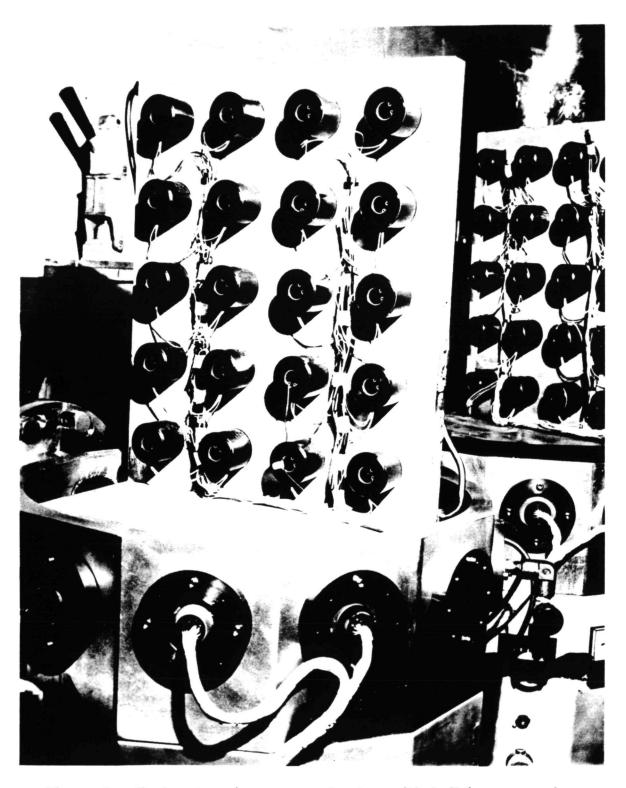


Figure 1. Test motors in vacuum chamber with bell jar removed.

I.D. by 0.625 in. O.D.), 440 C steel (RC 60-65) with ribbon type stainless steel cages. An approximate 10 to 15 percent fill of the candidate greases is applied to each bearing.

The motors used in these tests have the following characteristics:

- a. Type ac hysteresis, single phase, 60 cycle
- b. Speed 3600 rpm, synchronous
- c. Current 0.22 amp.

Because these motors do not use brushes, no problems are encountered with brush dust contamination of the bearings. In addition, these motors use approximately the same current when stalled as when operating at 3600 rpm; consequently, a bearing failure does not cause motor damage from overheating. A disassembled motor bearing set is shown in Figure 2.

To control temperature, the motors are mounted in an aluminum plate which is furnished with passages so that thermal control fluids (water or liquid nitrogen) may be used to control the motor temperature. Temperature is measured by thermocouples attached to the mounting plate and to selected motor cases.

Each mounting plate with its motor set is placed in a glass bell jar vacuum system. These bell jars are part of a 12-position vacuum system which is capable of maintaining pressures in the 1.3 \times 10 $^{-4}$ N/m 2 (1 \times 10 $^{-6}$ torr) range during test operation. The same bell jars are used for the oxidation and low temperature tests.

HI. TEST PROCEDURE

Since most bearings operating in space are not subject to a radial load, the major load to the test bearings is a thrust load applied by a wave washer. The motors, specially ordered from the manufacturer, are shimmed to maintain a $2.27~\mathrm{kg}$ (5 lb) thrust load on both bearings. This is equivalent to a $1.28 \times 10^9~\mathrm{N/m}^2$ (185 000 psi) Hz stress on the balls and inner races. The 3600 rom speed allows 216 000 rev/h on each bearing until failure. Each bearing which survives the 1-year test will have completed approximately 1 892 000 000 revolutions.

At the beginning of the test program, 25 lubricants from seven general chemical classes were selected for evaluation, with 13 lubricants being added after the test program had begun. These lubricants were selected to represent most of the military grease specifications, as well

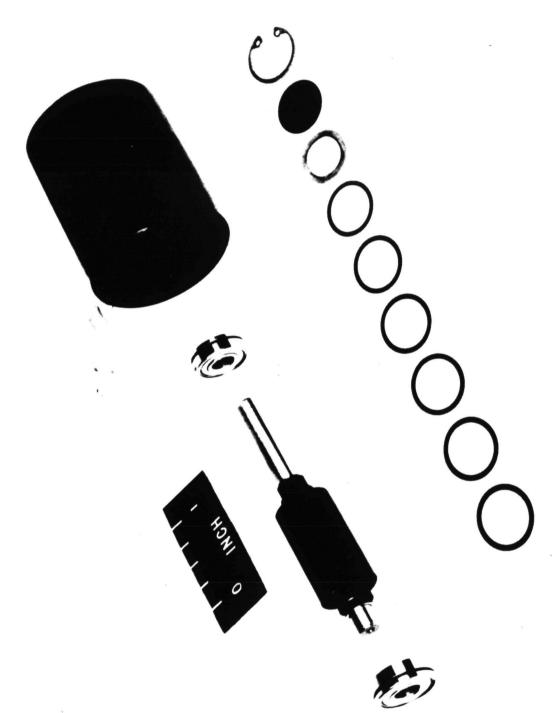


Figure 2. Disassembled ac motor with R-4 bearings.

as special nonspecification materials which had shown promise in space applications. The code designations given do not necessarily indicate different chemical compositions; the greases designated PFPE-4, PFPE-5, and PFPE-6 are from the same supplier, but with different base oil viscosities.

A general description of these greases is given in Table 1. It is planned to add additional lubricants to the test program (13 lubricants have been added since the start of the program) if data on new lubricants indicate that they have characteristics that would make them good candidates for one or more of the environments being used in the test program.

The environments for the test program to date are as follows:

- a. 6.895 \times 10 4 N/m 2 (10 psi) $\rm O_2$ at 90 percent relative humidity (oxidation tests)
 - b. Vacuum, ambient temperature (38°C)
 - c. Vacuum, high temperature (93.3°C)
 - d. Vacuum, ambient temperature, with start-stop operation
 - e. Low temperature start.

The present status of the test program is given in Table 2.

The evaluations for all tests, except the low temperature tests, are based primarily on a go/no-go system. The motor torque is low and the inertia of the system is low; therefore, when the bearing tends to seize, the motor stops without further damage to the bearings. The following data are taken during the test:

- a. Total test time
- b. Vacuum or atmospheric conditions
- c. Temperature
- d. Total cycles, if appropriate.

The bearings are weighed before and after testing, and the percent of weight loss of lubricant is calculated. The bearings are then photographed and cleaned, and selected bearings are subjected to scanning electron microscope (SEM) examination. Chemical analysis is made where applicable. SEM's and chemical analysis have not been added to this report.

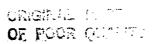


TABLE 1. DESCRIPTION OF TEST LUBRICANTS

Manufacturer Designation	Lubricant Code	MIL Spec	Gen. Chem. Class of Base Oil	Thickener	38C Oil Viscosity (cs)	Oil Viscosity Index	Description of Greases
KG 80	M-1 N-2	83176	Highly Refined Mineral	Inorganic	158	101	Instrument Brg.
Aeroshell 5	N-3	3545B	Mineral	Microgel	300	011	Hi Temp Acft
Royco 24R	N-4	10924B	Mineral	Li Soap			General Purpose
Royco 49	M-5	23549.A	Mineral	MoS2-Nonsoap			General Purpose
Royco 19B	,	23549B	Mineral	MoS2-Nonsoap			General Purpose
Aeroshell 14	N-6	25537A	Mineral	Ca Soap	+-	-	Ocsillating Brg.
Aeroshell 16	- N	25760A	Synthetic Mineral	Microgel	38		Brg., Wide Temp. Range
Thitemn 500	0 - N		Mineral-Diester	None Na Soan	10		Vacuum Brg Wido Tomp Range
Nobilgrease 28	7-10	81322	Synthetic Hydrocarbon	Nonsoap	}		Hi Temp. Acft.
Conoco HD ≈2	N-11		Mineral	Synthetic	108		Hi Temp, Corr. Resistant
BP 2110	N-12		Mineral	Graphite-Lead	119.7	107	Brg Vacuum
Exxon Andok C	N-13		Mineral	Na Soap	110		Long Life Anti-Friction
Supermil 06752	ES-1	25760A	Diester	Arylurea			Brg., Wide Temp.
Aeroshell 17	ES-2	21164C	Diester	Microgel	-		Wide Temp. with Mos,
Aeroshell	ES-3	23821.3	Diester	Microgel	7.		Acft. Instrument
L-11G	ES-4			Li Soap + Mos.	1+		Acft. Instrument
Exxon 3182	יוני הייני	10000		in Soap	291	190	Hi Temp. Acit.
Beacon 323	0 1 -	7307	Synthetic Ester	L. Soap	9		Low Temp.
DC No. 33			Silicone	Li Soan	150	5	Vector
G-351	Si-3		Silicone	Li Sono	2	•	Hi Tenn. Ball Bre.
Supermil 31052	Si-3	25013D	Silicone	Organic Dye			Ball and Roller Brg.
G-330M	Si-1		Silicone	•		-	General Purpose
G-3411.	0 10 10		Suicone				Acft, and Instrument
31.21.2	S1-N		Silicone	Silica			Rad. Res't. Brg. Experimental
FS-1290	ES-23		Fluorossilicone	Suica			Vac. Low Speed Brg.
Kei-F No. 90	FCC-1	_	Fluoro-Carbon	Silica			Chem Inch Der.
	PFPE-1		PFPE	Fluorotelomer	151	12.9	Hi Vac Brd
31 38RP	PFPE-2		PFPE	Fluorotelomer	129	350	Chem. Incrt Hi and Low Tenn.
3L-38RP Baked*			PFPE	Fluorotelomer			
631A	PFPE-3		PFPE	Fluorotelomer	153	110	Chem. Incrt Brg.
Z40.4Z	PrPE-4		3d4d	Fluorotelomer	18	23	Chem. Inert Low Temp.
240.48	PrPE-5		PFPE	Fluorotelomer	85	113	Chem. Inert Vacuum. Hi Temp.
31. 96. 16	Prre-b		7 7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	Fluorotelomer	270	134	
SE-38-15	FFFE-		PFPE	Fluorotelomer			Chem. Inert Wide Temp, with MoS,

*Vacuum baked at 100°C (212°F) for 20 hours.

TABLE 2. PRESENT STATUS OF LUBRICANT TESTS

			Тс	est Conditio	ns	
Lube Cod	e	Oxidizing Environment	Vacuum (38°C)	Vacuum (93, 3°C)	Vacuum (Start- Stop)	Low Temperature Start
KG 80	M-1	a	a	:t	a	а
SRG 200	M - 2	1	a	а	a	a
Aeroshell 5	M - 3		a,e	a,c	a,c	а
Royeo 24R	M - 4		a			а
Royco 49	M 5		a ·	a	a	a
Royco 49B			ļ		b	
Acroshell 14	M - 6		a			а
Aeroshell 16	M 7		a			
Apiezon L	M 8		a			
Unitemp 500	M 9		a			
Mobilgrease 28	M · 10		a	a	a	
Conoco HD#2	M 11		a	i)	a	a
BP 2110	M 12		a	ь	b	a
Andok C	M 13		a	ь	b	a
Supermil 06752	ES 1	a	a		a	a
Aeroshell 17	ES 2		a			
Aeroshell 7	ES 3			a	a	a
L-11G	ES 4		a			я
Exxon 5182	ES 5		a	a	ä	a
Exxon 325	ES 6		a	b		a
BP 8135	ES 7		a	ь	b	
DC No. 33	Si 1	វា	a			
G- 351	Si 2	a	a,e	a.e	a,e	a
Supermil 31052	Si 3		a	a	a	a
G 330M	Si 4		a	a		a
G 341L	Si 5		a	ล	н	a
31.27 2	Si X	a	:1	;1		
FS 1281	FS 1	a	a	Ì		
FS 1290	FS 2		a	a,a	a	
Kel F No. 90	FCC 1			a		
803	PFPE 1	a ,a	a,e	a,e	a .e	a
31 38RP	PFPE 2	a	a,c	a,a,e	a .e	a
3L 38RP Baked*			ង	a	a	a
631 A	PFPE 3		a	a	а	a
240Az	PFPE 4	a l	a	:1	а	a
240AB	PFPE 5		a	a	n	
240AC	PFPE 6		a	a	a	at .
3L 38 MS	PFPE~7		a	ь	ь	a

a Test complete, 1 year or two days (low temperature test only)
 b Test underway, 1 year
 c Test underway, 5 year

^{*}Vacuum baked at 100°C for 20 hr.

In the low temperature tests, the motors are installed in the cooling plate, and the system is evacuated to prevent frost formation. LN_2 is circulated through the cooling plate. The temperature is measured with thermocouples in contact with the outer race of the front bearing. Before cooling is initiated, the motors are operated for 30 min to channel the grease. The temperature is then dropped to -100°C and held approximately 30 min. The temperature is then allowed to rise slowly using a thermocouple on the mounting plate for control. After each 3°C rise, the motors are switched on for approximately 5 s, and the temperature of the front bearings is recorded. When each motor starts and comes up to full speed, the front bearing temperature is used as the low temperature starting capability of the lubricant. The starting torque of the motors used in this test is 1.05×10^{-2} N m (1.5 in. oz). Each low temperature test is repeated at least twice, and an average temperature is taken of the four motors and two tests.

IV. TEST RESULTS

A. Low Temperature Start Tests

At the present time, 26 of the candidate lubricants have been evaluated for low temperature capability. Unfortunately, the temperature at which the bearings will stall is a function of the volume of grease in the bearing, as well as the viscosity of the grease; therefore, some variation in stall temperature is sure to occur. To help overcome this difficulty, four motors are tested with each lubricant and at least two tests are made on each motor. The resulting stall temperatures are then averaged. Results of these tests are shown in Table 3. Ordinarily the vacuum stability requirements and the low temperature starting torque requirements are mutually exclusive because a low viscosity fluid provides better low temperature capabilities and a high viscosity fluid tends to be more vacuum stable. The results of these tests are, therefore, rather surprising since the PFPE-2 grease, which has a 38°C viscosity of 130 cs, has superior low temperature capabilities and is also one of the most vacuum stable greases evaluated. These capabilities are somewhat more understandable when it is noted that the base oil for this grease has a viscosity index of 350 and a molecular weight of over 9000.

B. Continuous Vacuum Ambient Temperature Tests

Ten 1-year tests have been completed; the results are given in Table 4. Sixty-four motors (16 lubricants) have had no failures resulting from lubricant depletion, but motor No. 3 of lubricant M-3 had a drive motor failure. Also, the first 13 lubricants listed have had less than a 20 percent average weight loss.

TABLE 3. LOW TEMPERATURE START, °C

Lubricant	1	2	3	4	Average
Si-3	-62.8	- 78. 9	-76.1	-70.0	-71.9
PFPE-7	-68.6	-68.6	-68.6	-68.6	-68.6
PFPE-2	-61.4	- 57. 5	-72.5	-82.2	-68.4
PFPE-2 Baked*	-68.1	-66.7	-64.7	-64.7	-66.0
M - 4	-58.9	-70.8	-60.0	-58.9	-62.1
M - 6	-56.7	-55.0	-60.3	-60.3	-58.1
ES-4	-53.9	-57.8	-55.8	-55.0	-55.6
ES-1	-51.1	-53.8	-51.1	-51.1	-51.8
Si-5	-49.2	-49.2	-49.2	-49.2	-49.2
ES-3	-53.9	- 41.1	-56.1	-42.1	-48.3
PFPE-1	-44.3	- 44.3	-49.4	-48.0	-46.5
ES-5	-42.5	-42.5	-46.4	-46.4	-44.5
ES-7	-43.6	- 42.8	-43.6	-43.6	-43.4
M-12	-42.8	-42.8	-42.8	-44.2	-43.2
ES-6	-41.4	-41.4	-41.4	-41.4	-41.4
PFPE-4	-36.1	- 36.1	- 36.1	- 36.7	- 36. 3
Si-4	-34.4	-34.4	- 34.4	-34.4	- 34. 4
M-13	-30.3	- 31.7	-30.3	-30.3	- 30.7
M - 5	-23.1	- 20.3	- 26.4	-21.1	-22.7
M-11	-21.9	-21.9	-21.9	-21.9	-21.9
Si-2	-16.7	-16.7	-16.1	-16.1	-16.4
M - 3	-16.1	-10.3	-16.1	-18.1	-15.2
M - 1	- 6.7	- 4.4	- 4.4	- 4.4	- 4.98
PFPE-6	- 4.4	- 4.4	+ 1.1	- 4.4	- 3.02
PFPE-3	~ 0.56	0.0	0.0	0.0	- 0.14
M-2	+ 3.30	+ 3.30	- 8.30	+ 3.30	+ 0.40

The average temperatures (10 tests) have been as follows:

Front bearing - 96°F (35.6°C) Rear bearing - 143°F (61.7°C) Mounting plate - 73°F (22.8°C).

C. Continuous Vacuum High Temperature Tests

Five 1-year tests have been completed; the results are given in Table 5. Forty motors (10 lubricants) have had no failures resulting from lubricant depletion, but motor No. 2 of lubricant M-2 had a drive motor failure. Also, the first seven lubricants listed have had less than a 20 percent average weight loss.

The temperature in these high temperature tests is controlled by regulating the cooling water supply to the mounting plate so as to maintain its temperature at 65°C (150°F). The average temperatures (five tests) have been as follows:

Front bearing - 172°F (77.8°C) Rear bearing - 208°F (97.8°C) Mounting plate - 154°F (67.4°C).

D. Continuous Oxidation Tests

During the development of the Skylab thermal control fan, problems were encountered with bearings operating in a highly oxidizing atmosphere; therefore, it was believed that a highly oxidative environment should form a part of the present evaluations.

TABLE 4. RESULTS OF VACUUM TESTS AT 38°C

		Но	urs to l	Failure ^a			Wei	ight Los	s (8) p	
Lubricant	1	2	3	4	Average	1	2	3	4	Average
PFPE-2	8760	8760	8760	8760	8760	5	7	8.5	5	6.5
Si-2	8760	8760	8760	8760	8760	3.5	12	6	4.5	6.5
M-5	8760	8760	8760	8760	8760	7.5	5	8	6.5	6.8
PFPE-2 ^d	8760	8760	8760	8760	8760	7.7	5.4	8.8	5.7	6.9
Si-4	8760	8760	8760	8760	8760	9.4	8.6	5.7	5.7	7.4
ES-5	8760	8760	8760	8760	8760	7.6	8.6	6.4	7.5	7.5
M-12	8760	8760	8760	8760	8760	6.5	12.4	12.6	6.1	9.4
PFPE-6	8760	8760	8760	8760	8760	6	13.5	12.5	7	9.8
M - 3	8760	8760	e	8760	8760	6	13	12	8.5	10
PFPE-3	8760	8760	8760	8760	8760	10	15.5	8.5	8	10.5
FS-2	8760	8760	8760	8760	8760	7	21	17.5	11.5	14
ES-7	8760	8760	8760	8760	8760	14.3	13.7	12	16.6	14.2
PFPE-1	8760	8760	8760	8760	8760	10.5	33	15	17	19
M-10	8760	8760	8760	8760	8760	26	20.5	19	23	22.1
M-13	8760	8760	8760	8760	8760	28	41.8	31.9	28.2	32.5
M - 2	8760	8760	8760	8760	8760	66	49	39	50	51
M-11	8513	8760	8760	8760	8698	20.1	19.6	15.4	22	19.3
Si-5	4739	8760	8760	8760	7755	9.5	5.4	11.4	3.1	7.4
PFPE-7	4397	8760	8760	8760	7669	27.2	6.0	2.5	2.3	9.5
M - 1	8760	8760	3700	8760	7495	21.5	27.5	23	25	24
Si-1	8760	8760	1709	8760	6997	35	25	41	22.5	31
PFPE-4	684	8760	8760	8760	6741	26	11.5	13	9.	15
ES-1	3524	8760	8437	4397	6280	24.5	39.5	23.5	18.5	26.5
M - 7	2530	8760	8760	3367	5854	53.5	47	54.5	42	49.5
PFPE-5	2096	3517	8760	8760	5783	33.5	40.5	3.5	3.5	20.3
Si-X	1041	6015	8760	5710	5382	27.5	28	40	47.5	36
M-8	392	8760	8524	1976	4913	3.3	0.8	0.8	11.3	4
ES-6	3563	5199	8760	1894	4854	61	67.8	59.6	68.3	64.2
M - 9	2543	1487	1199	8760	3497	34.5	27.5	49.5	24.5	34
Si-3	5613	2164	1659	456	2473	52.5	27	43.5	24.5	36.9
M - 4	2671	859	311	160	1000	74.5	73.5	82	78	77
ES-2	427	696	743	911	694	61.5	56	72.5	62	63.5
ES-4	559	593	559	823	634	30.5	32.5	39	41	35.5
FS-1	174	245	831	511	440	7.5	14.5	22.5	15.5	15
M-6	473	219	336	286	329	67	76	68.5	70.5	70.5

a. Or to end of test (1 year = 8760 hr).

Percent of weight loss of total weight of grease added to the two bearings of each motor (motor Nos. 1 through 4).

c. Drive motor failed.

d. Baked in vacuum at 100°C for 20 hr.

RESULTS OF VACUUM TESTS AT 93.3°C TABLE 5.

		Hour	s to	Failure ^a			Wei	Weight Loss	q(%) s	
Lubricant	. 1	2	3	4	Average	-	2	3	4	Average
FPE-2	8760	8760	<u></u>	~	~	13		14	17	14.5
FPE-	8760	8760	\sim	~	<u></u>	₹.	14.2	2		15
PFPE-6	8760	8760	8760	8760	8760	19.5	6	19.5	13.5	15.5
FPE-	8760	8760	[~	1	~	14		12	2.	16
FPE-	8760	8760	<u></u>	<u>_</u>	~	18		24.5	12	17
	8760	8760	(~~	1-	<u></u>	15	24.5	4.	15.5	7
ш	8760	8760	Ĺ-	[-	~	18		24	19	6
M-3	8760	8760	[:-	[~	~	29.5	35	27	34.5	31.5
- 1	8760	8760	Ĺ~	~	~	29	37	32	43	ت
- 1	8760	ပ	l —	[-	L~	55	31	50	47.5	46
FS-2	6813	8760	[-	[-	\sim	59	35.5	0	35	40.5
٠	8750	2870	[-	<u></u>	\sim	23	51	23.5	36	ლ
1	0928	5658	4	\sim	77	4.	41.7	ъ.	43.6	36
	1218	8760	g	9			6	27	25	₹.
Si-5	8760	755	515	[-	9	6.7	11.8	3		10.4
ES-5	2432		4	(L)	٠Ţ٠	œ.		40.8	34.9	ъ.
Si-3	989	2290	1702	(L)	[~	_	41	∞ .	2	د
PFPE-4	3193	350	വ	282	വ	54	39	63	44	20
M-10	1091		\sim	1274	4	68.7	73.8	48.3	63.3	ь Э
FCC-1	353	1280	521	166	580	47	53	2	54	0
$\dot{\gamma}$	174	101	1047	68.5	348	70.5	59.5	26	62.5	62.5
ES-3	82	73	20	71	74	٠.		83.5	88	7.

Or to end of test (1 year = 8760 hr). Percent of weight loss of total weight of grease added to the two bearings of each motor (motor Nos. 1 through 4). Drive motor failed. Baked in vacuum at 100°C for 20 hr. ъ. С

g.,

The first set of tests was made in air at 90 percent relative humidity. However, it appeared that a pure oxygen environment would be more severe; therefore, an additional set of tests was made in 10 psi pure oxygen at 90 percent relative humidity. Although no temperature measurements were made during these tests, bearing operating temperatures should have been very close to those reported for the 38°C tests, since the operating procedure for controlling cooling water flow to the motor mounting plate was essentially the same.

Only the two 1-year tests previously mentioned have been completed; the results are given in Table 6. Thirty-two motors (eight lubricants) have had no failures resulting from lubricant depletion, but motor No. 3 of lubricant Si-1 had a drive motor failure. Also, the first five lubricants listed have had less than a 20 percent average weight loss.

E. Start-Stop Vacuum Ambient Temperature Tests

Since many mechanisms do not operate continuously, it was decided to simulate the boundary conditions which exist between the balls and races of a bearing during acceleration and deceleration. Timers are used to shut off the motors for 10 s every 150 s (24 c/h) or for 20 s every 180 s (20 c/h).

Four 1-year tests have been completed; the results are given in Table 7. Thirty-six motors (nine lubricants) have had no failures resulting from lubricant depletion. Also, the first seven lubricants listed have had less than a 20 percent average weight loss.

Cycle counters are used at the start-stop stations to record the total number of cycles. The total cycles of the four completed tests were as follows:

- 1. 202 382
- 2. 188 342
- 3. 175 206
- 4. 177 337.

The average temperatures (four tests) were as follows:

Front bearing - 96°F (35.6°C) Rear bearing - 114°F (45.6°C) Mounting plate - 70°F (21.1°C).

V. FUTURE PLANS

Since all but four lubricants have been eliminated for the 5-year test program, a rating sheet (Table 8) was devised to eliminate those lubricants which perform poorly under the various test environments.

TABLE 6. RESULTS OF OXIDIZING TESTS

		Hour	Hours to Failure	ilure			Weig	Weight Loss (%) ^b	(%)	
Lubricant	1	2	3	4	Average	1	2	3	4	Average
Si-2	8760	8760	8760	8760	8760	9.6	1.7	4.0	3.7	4.8
PFPE-1c	8760	8760	8760	8760	8760	ည	5.5	5.5	ເລ	5.3
PFPE-1	8760	8760	8760	8760	8760	6.7	3.8	20.8	8.8	10.0
ES-1 _C	8760	8760	8760	8760	8760	12.5	12	11.5	12	12
M-1	8760	8760	8760	8760	8760	20	19	17.8	22.6	19.9
Si-X ^c	8760	8760	8760	8760	8760	35.5	40.5	43	42	40
PFPE-4	8760	8760	8760	8760	8760	50.9	30	70.7	39.0	47.7
Si-1 ^c	8760	8760	Ö	8760	8760	48.5	17	70	46	45.4
PFPE-2	8760	4795	8760	8760	6922	6.7	47.1	11.8	11.3	19.2
FS-1 ^c	8760	405	8760	8760	6671	က		က	4.5	3.5

Or to end of test (1 year = 8760 hr).

Percent of weight loss of total weight of grease added to the two bearings of each motor (motor Nos. 1 through 4). ъ. О

These tests were run in air at 90% relative humidity. All other tests were run in 10 psi pure oxygen at 90% relative humidity. ပံ

Drive motor failed. p

TABLE 7. RESULTS OF START-STOP TESTS

		Hour	Hours to Failure	lure			Weigh	Weight Loss (%	q(%)		Cycle
Lubricant	1	2	3	4	Average	1	2	3	4	Average	(s)
PFPE-6	8760	8760	8760	8760	8760	80	သ	6.5	4	5.4	180
PFPE-1	8760	8760	8760	8760	8760	4.5	2	3.5	10	5.8	150
PFPE-2c	8760	8760	8760	8760	8760	5.7	6.1	6.7	6.1	6.2	180
PFPE-2	8760	8760	8760	8760	8760	7	8.5	4.5	7.5	7	150
ES-5	8760	8760	8760	8760	8760	7.1	7.4	8.8	5.2	7.1	180
M-3	8760	8760	8760	8760	8760	12	9	8. n	10.5	9.3	180
PFPE-3	8760	8760	8760	8760	8760	6.9	20.5	12	.82	18	180
M-11	8760	8760	8760	8760	8760	24.3	21.1	20.6	17.3	20.8	180
M-5	8760	8760	8760	8760	8760	23	31	12	25	22.8	180
M-1	8760	8760	6790	8760	8268	6.5	14	36.5	11	17	180
Si-3	5409	8760	8760	8760	7922	32	12.5	12	14	17.5	180
M-10	6261	6313	8760	8760	7.52.4	다. 1-1	46.6	22.8	37.1	38.3	180
PFPE-5	8760	8760	8760	2817	7274	10	(G 27	1.5	ري دي	· •	150
ES-1	5783	8760	5497	8760	7200	44.0	91	55	15	33.5	180
M-2	8760	8760	1848	8760	7032	2.2	56	46	20	e 0€	180
Si-2	8760	1557	8760	8760	69 29	ಣ	9.5	2	ច	ວ. ວ	150
FS-2	685	8760	8760	5684	5972	15	19	21.5	25.5	20.5	150
Si-5	2222	8760	629	8760	5932	40.3	5.1	31.8	8.5	21.4	180
PFPE-4	4977	4737	5926	6586	5557	84	76.5	66.5	20	74.3	180
ES-3	3345	3501	2117	4340	3326	92	69.5	68.5	89	70.5	180

Or to end of test (1 year = 8760 hr).

Percent of weight loss of total weight of grease added to the two bearings of each motor (motor Nos. 1 through 4). ъ. С

Baked in vacuum at 100°C for 20 hr. ပ

The ratings are made by assigning the number 1 to the lubricant which performs the best in a particular test, the number 2 to the second best, etc. Where several lubricants are considered equal, the positions are averaged and assigned to all of the equivalent lubricants. Table 8 is used to illustrate the comparative principle only, because some of the tests are not complete and some of the greases have not yet been tested; however, using this chart, it was decided to eliminate 15 of the materials from further testing because they have performed poorly in either the vacuum ambient or vacuum high temperature tests.

The present test program will continue with the four candidate lubricants for the 5-year test program. Since the last status report, five additional lubricants have been placed under test in selected environments for 1-year periods to see whether these lubricants warrant further testing. As many samples of each grease as possible will be evaluated.

VI. PRESENT STATUS

One hundred tests are now underway, and the status of these tests as of December 1980 is shown in Table 9. The present test series is now progressing rapidly with three 5-year tests and two 1-year tests in operation.

VII. CONCLUSIONS

Some testing remains to be done in this program; however, from the data so far the following conclusions from the vacuum tests are being made:

- a. As a whole, the chemical class listed as PFPE in Table 1 has given the best results in all the vacuum tests completed to date.
- b. In the vacuum ambient temperature tests, PFPE-2 (as manufactured and vacuum baked) and PFPE-6, Si 2 and Si-4, M-5 and M-12; and ES-5 have given the best results with less than a 10 percent average weight loss.
- e. In the vacuum high temperature tests, M-5 and all the PFPE greases, except PFPE-4 and PFPE-7, have given the best results with less than a 20 percent average weight loss. Note from Table 9 that testing of PFPE-7 grease is incomplete.
- d. In the start stop tests, ES 5, M 3, and the PFPE greases (except PFPE 3, PFPE 4, and PFPE 5) have given the best results with less than a 10 percent average weight loss.

TABLE 8. LUBRICANT RATING CHART

Lube Cod	e	Oxidizing Environment	Vacuum (38°C)	Vacuum (93.3°C)	Vacuum Start-Stop	I.ow Temperature Start	Decision (See Note)
KG80	M - 1	4.5	20	5.5	10	23	
SRG 200	M-2		8.5	5.5	15	26	1
Aeroshell 5	M - 3		8.5	5.5	5	22	ĺ
Royco 24R	M - 4		31			5	EL
Royco 49	M - 5		8.5	5.5	5	19	
Royco 49B				٠		٠,	
Aeroshell 14	M - 6		35			6	EL
Aeroshell 16	M - 7		24				EL
Apiezon L	M - 8		27				EL
Unitemp 500	M - 9		29				EL
Mobilgrease 28	M-10		8.5	19	12		
Conoco HD#2	M-11		17	13	5	20	
BP 2110	M - 12		8,5			14	
Exxon Andok C	M - 13		8.5			18	
Supermil 06752	ES-1	4.5	23		14	8	EL
Aeroshell 17	ES-2		32			_	EL
Aeroshell 7	ES - 3			22	20	10	EL
L-11G	ES-4		33			7	EL
Exxon 5182	ES 5	İ	8.5	16	5	12	
Beacon 325	ES-6		28			15	
BP 8135	ES - 7		8.5			13	
DC No. 33	Si·1	4.5	21				EL
C - 351	Si - 2	4.5	8.5	12	16	21	
Supermil 31052	Si · 3		30	17	11	i	EL
G-330M	Si-4		8.5	14		17	EL
G-341L	Si-5		18	15	18	9	
31,27-2	Si-X	4.5	26	21		-	EL
FS-1281	FS 1	10	34				EL
FS-1290	FS 2		8.5	11	17		
Kel-F No. 90	FCC-1			20			EL.
803	PFPE-1	4.511	8.5	5, 5	5	11	
3L - 38RP	PFPE 2	9	8.5	5,5	5	3	
3L 38RP Baked	· · · · · · · · · · · · · · · · ·	"	8.5	5.5	5	4	
631A	PFPE 3		8, 5	5.5	5	25	1
240AZ	PFPE 4	4.5	22	18	19	16	
240AB	PFPE - 5	"	25	5,5	13	• • •	
240AC	PFPE-6		8.5	5.5	5	24	
3L - 38: MS	PFPE 7	ľ	19	1	•	2	

Note: EL - eliminate from further testing.

a. Two tests (see Table 2).

TABLE 9. HOURS TO FAILURE IN TESTS NOW OPERATING

Cont	tinuous Va	cuum A	mbient	Tempe	rat	ure (:	started 6-2:	3-76) 5 year
	PFPE-1	31 91	.8 22	676			21 140	32 173
	PFPE-2	•					•	
	M-3 Si-2	10 22	23 21	191	20	006		1 411
	51-2	19 32	3 41	444	32	000		1 411
Star	t-Stop Va	cuum A	mbient 1	Гетрез	ratu	ıre (s	started 11-1	2-76) 5 year
	PFPE-1			978			11 116	
1	PFPE-2			0.0			5 275	
1	M-3				11	510	4 901	7 006
Ì	Si-2				* *	658	29 133	586
Conf	tinuous Va	ecuum 1	ligh Ten	nperat	ure	(star	ted 4-18-78	3) 5 year
}	PFPE-1		3	971	5	754	9 012	
	PFPE-2	2 17			_			
	M-3						19 557	
Ι.	Si-2	17 87	7		1	759	21 393	20 277
Cont	tinuous Va	icuum E	ligh Ten	perat	ure	(star	ted 4-28-80) 1 year
	M-12		1	745				
	ES-7	16		57		125	177	
1	PFPE-7	2 07		057		123	7.1.4	
	M-13	1 90		673	1	362		
	ES-6	1 03		761	,	729	594	
	110 0	1 00	,, ,	101		123	334	
Star	t-Stop Va	cuum A	mbient 1	remper	atu	re (s	started 10-2	8-80) 1 year
	M-12							
	ES - 7							
	PFPE-7							
	M-13							
	M - 5							
	171 0			<u> </u>				

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APPROVAL

AN EVALUATION OF GREASE TYPE BALL BEARING LUBRICANTS OPERATING IN VARIOUS ENVIRONMENTS (Status Report No. 5)

By E. L. McMurtrey

The information in this report has been reviewed for security classification. Review of any information concerning Department of Defense or nuclear energy activities or programs has been made by the MSFC Security Classification Officer. This report, in its entirety, has been determined to be unclassified.

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